

Advanced System Dynamics and Control
2.151
Spring 2001
Solution Homework 6

Massachusetts Institute of Technology
Department of Mechanical Engineering

Solution 1 : Transfer functions.

(a) The transfer functions are

$$\begin{aligned}
 \text{Forward loop TF: } G(s)G_c(s) &= \frac{\beta k_d s + \beta k_p}{s^2 - \alpha} \\
 &= \frac{888.425s + 426035}{s^2 - 31250} \\
 \text{Return difference TF: } 1 + G(s)G_c(s) &= \frac{s^2 + \beta k_d s + \beta k_p - \alpha}{s^2 - \alpha} \\
 &= \frac{s^2 + 888.425s + 394785}{s^2 - 31250} \\
 \text{Sensitivity TF: } \frac{1}{1 + G(s)G_c(s)} &= \frac{s^2 - \alpha}{s^2 + \beta k_d s + \beta k_p - \alpha} \\
 &= \frac{s^2 - 31250}{s^2 + 888.425s + 394785} \\
 \text{Closed loop TF: } \frac{G(s)G_c(s)}{1 + G(s)G_c(s)} &= \frac{\beta k_d s + \beta k_p}{s^2 + \beta k_d s + \beta k_p - \alpha} \\
 &= \frac{888.425s + 426035}{s^2 + 888.425s + 394785}
 \end{aligned}$$

- (b) (i) **Forward loop transfer function:** This transfer function has two poles and one zero. The DC gain is 13.6331. The high frequency slope is -20dB/decade . The phase starts at -180° at low frequencies and reaches -90° at high frequencies. The phase increases due to the zero. The forward loop transfer function bode plot is shown in Figure 1(a).
- (ii) **Return difference transfer function:** This transfer function has 2 poles and 2 zeros. The gain is constant in the low and high frequency ranges. The -40dB/decade is contributed by the poles and the $+40\text{dB/decade}$ by the zeros. The phase is -180° at low frequencies because the poles contribute -180° (due to the unstable pole) and there is no phase contribution by the zeros in this range. At high frequencies, the 2 zeros contribute $+180^\circ$. This cancels out the -180° contribution of the poles resulting in a zero phase. The return difference transfer function bode plot is shown in Figure 1(b).
- (iii) **Sensitivity transfer function:** This transfer function has 2 poles and 2 zeros. It is the inverse of the return difference transfer function. The phase is $+180^\circ$ in the low frequency

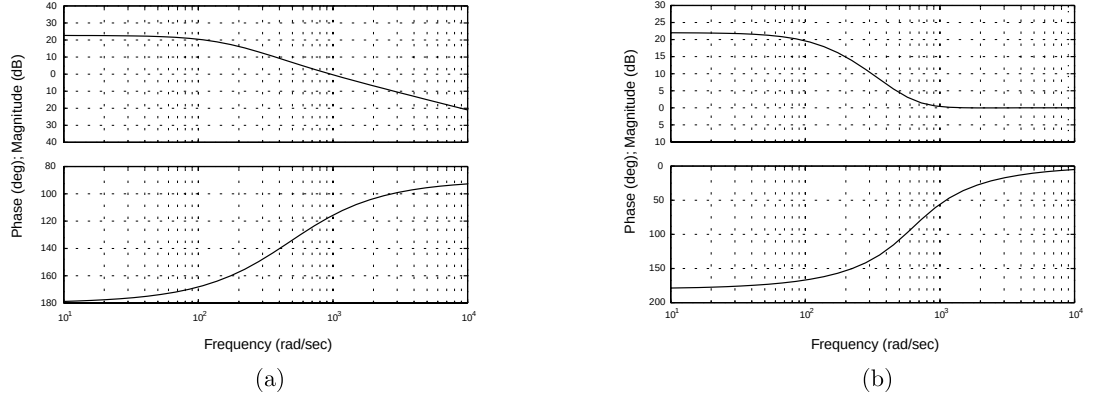


Figure 1: Magnitude and phase plots. (a) Forward loop transfer function. (b) Return difference transfer function.

range and 0 in the high frequency range. The sensitivity transfer function bode plot is shown in Figure 2(a).

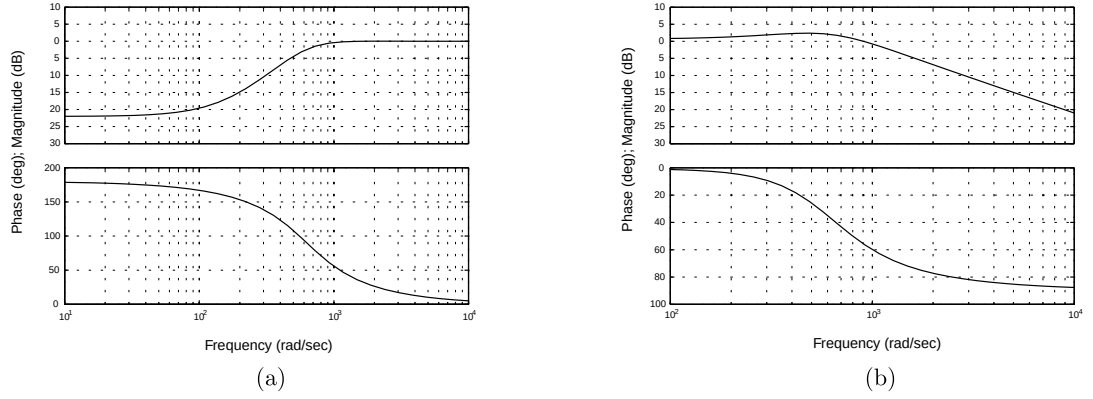


Figure 2: Magnitude and phase plots (a) Sensitivity transfer function. (b) Closed loop transfer function.

- (iv) **Closed-loop transfer function:** The closed loop transfer function has 2 poles and 1 zero. The magnitude plot is constant in the low frequency range and decreases at -20 dB/decade in the high frequency range. The phase on the other hand is 0° in the low frequency range and -90 in the high frequency range. The closed-loop transfer function bode plot is shown in Figure 2(b).
- (c) The system bandwidth is the frequency range $0 \leq \frac{\omega}{2\pi} \leq f_b$ where f_b is the frequency at which the gain has dropped by 3 dB below the DC gain. If we consider the forward loop transfer function $G(s)G_c(s)$, its gain must be high in the system bandwidth to ensure good command following and disturbance rejection. From the forward loop transfer function bode plot, $f_b = 119.3668$ rad/s $= 18.9987$ Hz.
- (d) The time response of the system to the given noise inputs are given in Figure 3. For the noise with frequency of $\frac{f_b}{2}$, the gain of the closed loop transfer function is approximately 1 dB, consequently, the noise is amplified by 1 dB. For the noise with frequency of $20f_b$, the gain of

the closed loop transfer function is approximately -10 dB, consequently, the noise is amplified by -10 dB.

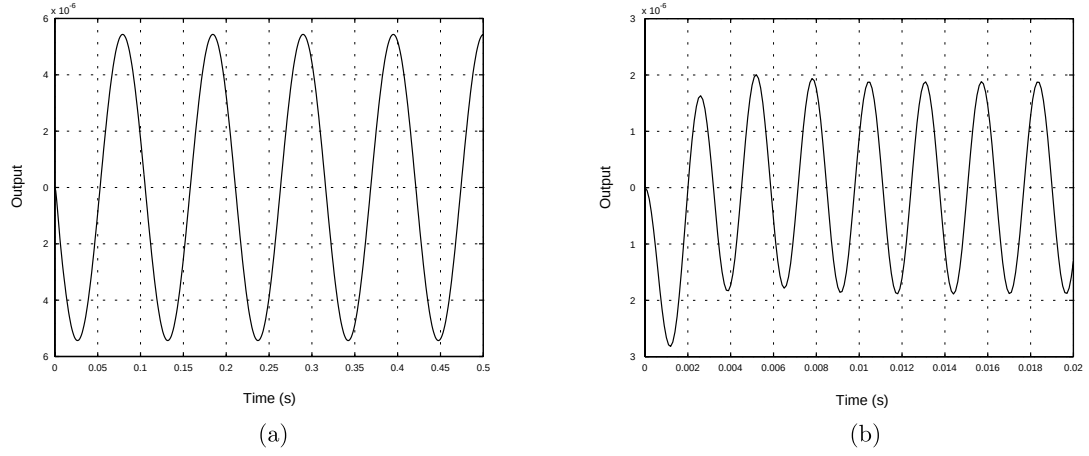


Figure 3: System time response to time inputs. (a) $N(s) = N_o \sin(\pi f_b t)$. (b) $N(s) = N_o \sin(40\pi f_b t)$

(e) The zero z_o introduced by the PD controller is found by

$$\begin{aligned} k_p + k_d s &= 0 \\ \Rightarrow s &= -\frac{k_p}{k_d} = -479.539 \text{ rad/s} \end{aligned}$$

The effects of the zero on the system response are shown in Figures 4 to 6. In this case, we have changed the location of the zero by changing k_p . Note that when the zero is closer to the origin, the time response slows down for this system.

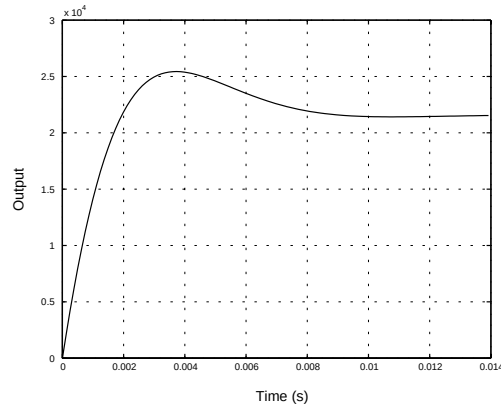


Figure 4: Effects of the zero on the system time response — original response.

Solution 2 : Transfer functions: A drive system. _____

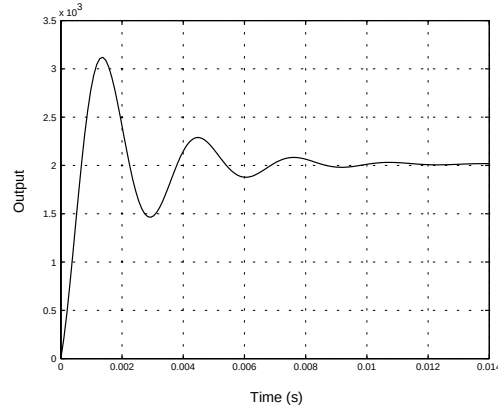


Figure 5: Effects of the zero on the system time response — $|z_1| = 10|z_o|$.

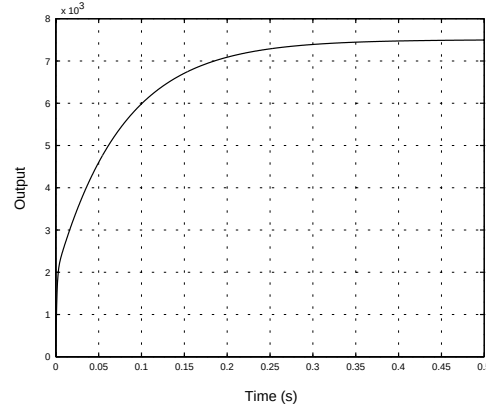


Figure 6: Effects of the zero on the system time response — $|z_2| = .1|z_o|$.

- (a) Using the given system parameters — gear ratio, $r = 0.5$, motor inertia $I_m = 0.08 \text{ ft.lb.s}^2.\text{rad}^{-1}$, load inertia $I_l = 0.8 \text{ ft.lb.s}^2.\text{rad}^{-1}$ and $R = 0.26 \text{ ft.lb.s.rad}^{-1}$, the effective system inertia I reflected on the motor axis is

$$\begin{aligned}
 I &= I_m + I_l r^2 \\
 &= 0.08 + 0.08 + 0.08(0.25) \\
 &= 0.28 \text{ ft.lb.s}^2.\text{rad}^{-1}
 \end{aligned} \tag{1}$$

- (b) The transfer function $\frac{\Omega(s)}{U(s)}$ is given by

$$\begin{aligned}
 \frac{\Omega(s)}{U(s)} &= \frac{1/I}{s + R/I} \\
 &= \frac{3.5714}{s + 0.9257}
 \end{aligned} \tag{2}$$

- (c) The magnitude and phase are shown in Figure 7. This is a first order system. At low frequencies, the output velocity will follow the command input with a small phase lag. The DC gain (the

gain at zero frequency) is $\frac{1/I}{R/I} = \frac{1}{R} = 11.7005 \text{ dB}$. For inputs with high frequencies, $\omega \gg \omega_b$, the output amplitude will drop significantly. In fact, it will behave as $|G(j\omega)| \approx \left| \frac{1/I}{j\omega} \right| = \frac{1}{I\omega}$. This is also equal to

$$\begin{aligned} 20 \log_{10} |G(j\omega)| &\approx -20 \log_{10}(I\omega) \\ &= -11.0568 - 10 \log_{10}(\omega) \end{aligned} \quad (3)$$

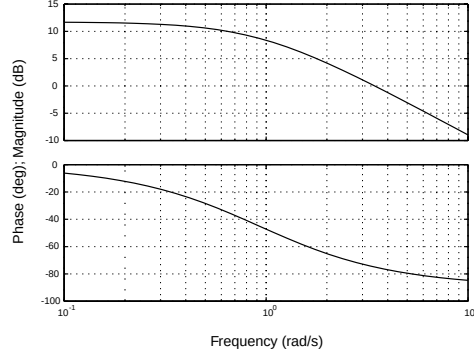


Figure 7: Magnitude and phase of $\frac{\Omega(s)}{U(s)}$.

On a logarithmic scale, the gain resembles a -20 dB/decade line. At high frequencies, the phase lag will be $\text{deg } -90$. This is shown by

$$\begin{aligned} \arg[G(j\omega)] &\approx \arg \left[\frac{1/I}{j\omega} \right] \\ &= \arg \left[\frac{-j}{I\omega} \right] \end{aligned} \quad (4)$$

The break frequency for this first order system occurs at the point where the phase is $\text{deg } -45$, i.e. $f_b = 0.1473 \text{ Hz}$.

- (d) The time response of the system for a step input of $u(t) = 95 \text{ ft.lb}$ is shown in Figure 8. The time constant is 1.077 s and the settling time is $4\tau = 4.308 \text{ s}$.
- (e) The time response of the system with sinusoidal inputs is shown in Figures 9 to 11. The frequency of the input $u_1(t) = 10 \sin(0.2\pi f_b t)$ is 10 times lower than the break frequency so the phase lag should be approximately $\text{deg } -6$ and the gain between the output and the input is approximately $|G(j\omega)| = R^{-1} = 3.858$. For our input amplitude of 10, the amplitude of the output should be ≈ 38.58 . The frequency of the input $u_2(t) = 10 \sin(2\pi f_b t)$ is equal to the break frequency, so the phase lag should be $\text{deg } -45$ and the gain between the output and the input is

$$\begin{aligned} |G(2\pi f_b j)| &= \left| \frac{1/I}{j\omega + R/I} \right| \\ &= 2.7241 \end{aligned} \quad (5)$$

The output amplitude should be approximately $2.741 \times 10 = 27.241$. The frequency of the input $u_3(t) = 10 \sin(20\pi f_b t)$ is 10 times higher than the break frequency, so the phase lag should be

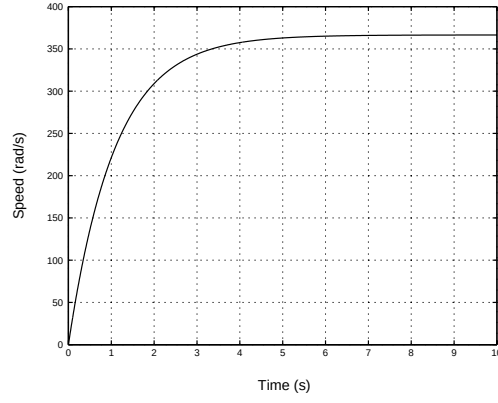


Figure 8: Time response of system for step input of $u(t) = 95 \text{ ft.lb.}$

approximately $\text{deg } 84$ and the gain between the output and input is

$$\begin{aligned} |G(20\pi f_b j)| &= \left| \frac{1/I}{I\omega} \right| \\ &= 0.3858 \end{aligned} \quad (6)$$

Thus, the output amplitude should be approximately $0.3858 \times 10 = 3.858$. These three cases indicate that the gain starts to decrease rapidly for frequencies higher than the break frequency.

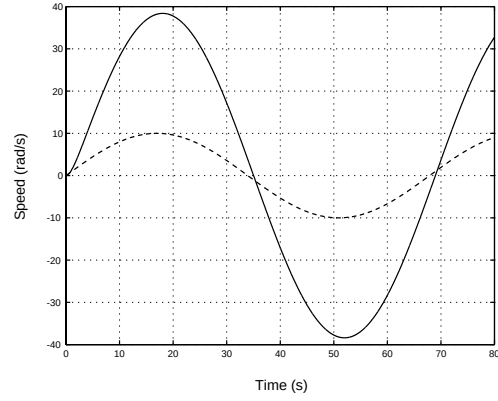


Figure 9: Time response of system for $u_1(t) = 10 \sin(0.2\pi f_b t)$.

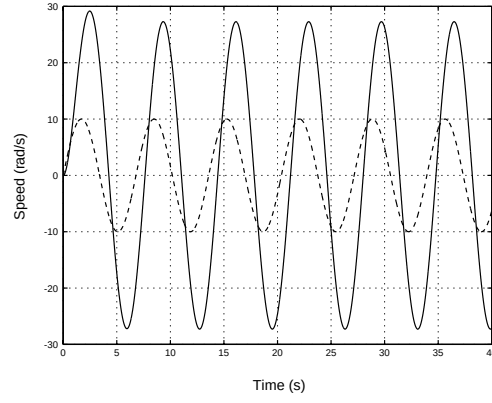


Figure 10: Time response of system for $u_2(t) = 10 \sin(2\pi f_b t)$.

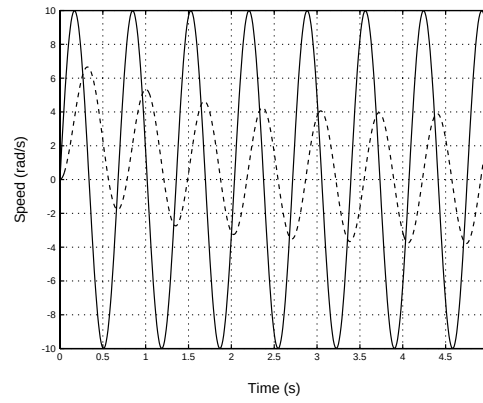


Figure 11: Time response of system for $u_1(t) = 10 \sin(20\pi f_b t)$.